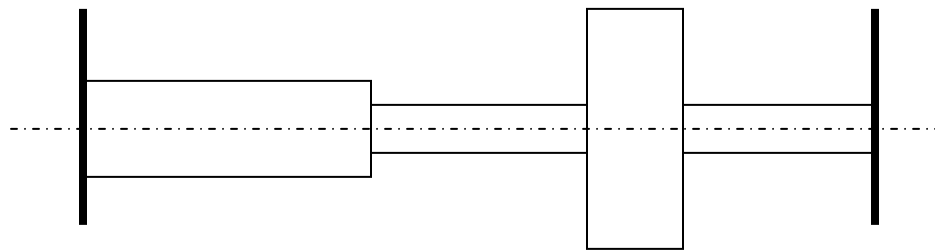


1. $\sin^2 \theta$ is equal to
 (a) 0 (b) \neq (c) 1 (d) -1
2. The accuracy of Simpson's rule quadrature for a step size h is
 (a) $O(h^2)$ (b) $O(h^3)$ (c) $O(h^4)$ (d) $O(h^5)$
3. For the matrix $\begin{bmatrix} 4 & 1 \\ 1 & 4 \end{bmatrix}$ the eigen values are
 (a) 3 and -3 (b) -3 and -5 (c) 3 and 5 (d) 5 and 0
4. The second moment of a circular area about the diameter is given by (D is the diameter).
 (a) $\frac{D^4}{4}$ (b) $\frac{D^4}{16}$ (c) $\frac{D^4}{32}$ (d) $\frac{D^4}{64}$
5. A concentrated load of P acts on a simply supported beam of span L at a distance $\frac{L}{3}$ from the left support. The bending moment at the point of application of the load is given by
 (a) $\frac{PL}{3}$ (b) $\frac{2PL}{3}$ (c) $\frac{PL}{9}$ (d) $\frac{2PL}{9}$
6. Two identical circular rods of same diameter and same length are subjected to same magnitude of axial tensile force. One of the rods is made out of mild steel having the modulus of elasticity of 206 Gpa. The other rod is made out of cast iron having the modulus of elasticity of 100 Gpa. Assume both the materials to be homogeneous and isotropic and the axial force causes the same amount of uniform stress in both the rods. The stresses developed are within the proportional limit of the respective materials. Which of the following observations is correct?
 (a) Both rods elongate by the same amount
 (b) Mild steel rod elongates more than the cast iron rod
 (c) Cast iron rod elongates more than the mild steel rod
 (d) As the stresses are equal strains are also equal in both the rods

7. Two beams, one having square cross section and another circular cross-section, are subjected to the same amount of bending moment. If the cross sectional area as well as the material of both the beams are the same then
- maximum bending stress developed in both the beams is the same
 - the circular beam experiences more bending stress than the square one
 - the square beam experiences more bending stress than the circular one
 - as the material is same both the beams will experience same deformation
8. The mechanism used in a shaping machine is
- a closed 4-bar chain having 4 revolute pairs
 - a closed 6-bar chain having 6 revolute pairs
 - a closed 4-bar chain having 2 revolute and 2 sliding pairs
 - an inversion of the single slider-crank chain
9. The lengths of the links of a 4-bar linkage with revolute pairs only are p, q, r and s units. Given that $p < q < r < s$. which of these links should be the fixed one, for obtaining a 'double crank' mechanism?
- link of length p
 - link of length q
 - link of length r
 - link of length s
10. Consider the arrangement shown in the figure below where J is the combined polar mass moment of inertia of the disc and the shafts. K_1, K_2, K_3 are the torsional stiffness of the respective shafts. The natural frequency of torsional oscillation of the disc is given by



(a) $\sqrt{\frac{1}{l_1} + \frac{2}{l_2} + \frac{3}{l_3}}$

(b) $\sqrt{\frac{1}{l_1} + \frac{2}{l_2} + \frac{3}{l_3}}$

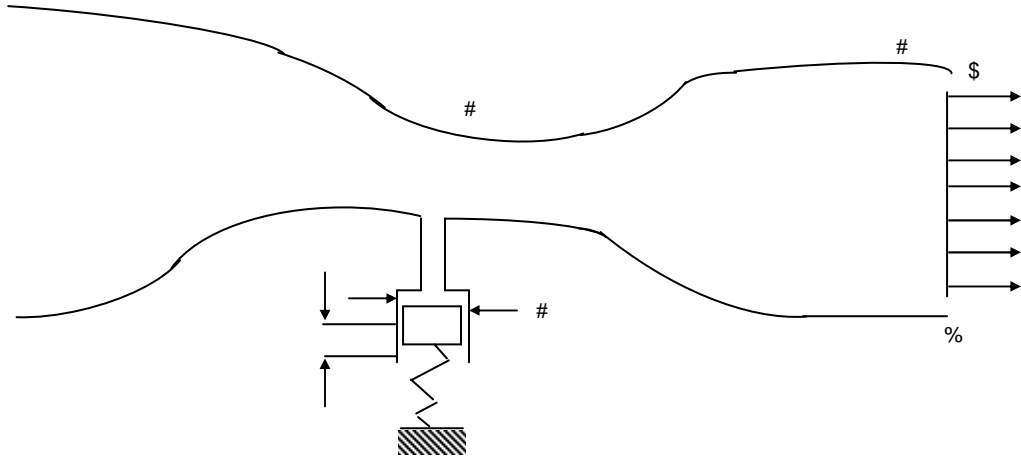
(c) $\sqrt{\frac{1}{l_1} + \frac{2}{l_2} + \frac{3}{l_3}}$

(d) $\sqrt{\frac{1}{l_1} + \frac{2}{l_2} + \frac{3}{l_3}}$

11. Maximum shear stress developed on the surface of a solid circular shaft under



47. Air flows through a venturi and into atmosphere. Air density is ρ ; atmospheric pressure is P_a ; throat diameter is D_t ; exit diameter is D and exit velocity is U . The throat is connected to a cylinder containing a frictionless piston attached to a spring. The spring constant is k . The bottom surface of the piston is exposed to atmosphere. Due to the flow, the piston moves by distance x . Assuming incompressible frictionless flow, x is



- (a) $(\rho Q U^2 / 2) p^{-2}$ (b) $(\rho U^2 / 8k) \frac{2}{2} - 1 p^{-2}$
 (c) $(\rho U^2 / 2k) \frac{2}{2} - 1 p^{-2}$ (d) $(\rho U^2 / 8k) \frac{4}{4} - 1 p^{-2}$

48. Consider a laminar boundary layer over a heated flat plate. The free stream velocity is U_∞ . At some distance x from the leading edge the velocity boundary layer thickness is d_v and the thermal boundary layer thickness is d . If the Prandtl number is greater than 1, then

- (a) $d_v > d$ (b) $d > d_v$
 (c) $d_v \gg d \sim (Pr)^{-1/2}$ (d) $d_v \gg d \sim Pr^{-1/2}$

49. Considering the relationship $TdS = dU + pdV$ between the entropy (S), internal energy (U), pressure (p), temperature (T) and volume (V), which of the following statements is correct?

- (a) It is applicable only for a reversible process
 (b) For an irreversible process, $TdS > dU + pdV$
 (c) It is valid only for an ideal gas
 (d) It is equivalent to 1st law, for a reversible process

50. In a gas turbine, hot combustion products with the specific heats $C_p = 0.98$ kJ/kgK, $C_v = 0.7538$ kJ/kg K enter the turbine at 20 bar, 1500 K and exit at 1 bar. The isentropic efficiency of the turbine is 0.94. The work developed by the turbine per kg of gas flow is
- (a) 689.64 kJ/kg (b) 794.66 kJ/kg
(c) 1009.72 kJ/kg (d) 1312.00 kJ/kg
51. An automobile engine operates at a fuel air ratio of 0.05, volumetric efficiency of 90% and indicated thermal efficiency of 30%. Given that the calorific value of the fuel is 45 MJ/kg and the density of air at intake is 1 kg/m³, the indicated mean effective pressure for the engine is
- (a) 6.075 bar (b) 6.75 bar (c) 67.5 bar (d) 243 bar
52. For an engine operating on air standard Otto cycle, the clearance volume is 10% of the swept volume. The specific heat ratio of air is 1.4. the air standard cycle efficiency is
- (a) 38.3% (b) 39.8% (c) 60.2% (d) 61.7%
53. A centrifugal pump running at 500 rpm and at its maximum efficiency is delivering a head of 30m at a flow rate of 60 litres per minute. If the rpm is changed to 1000, then the head H in metres and flow rate Q in litres per minute at maximum efficiency are estimated to be
- (a) $H = 60, Q = 120$ (b) $H = 120, Q = 120$
(c) $H = 60, Q = 480$ (d) $H = 120, Q = 30$
54. Hardness of steel greatly improves with
- (a) annealing (b) cyaniding (c) normalizing (d) tempering
55. With a solidification factor of 0.97×10^6 s/m², the solidification time (in seconds) for a spherical casting of 200 mm diameter is
- (a) 539 (b) 1078 (c) 4311 (d) 3233
56. A shell of 100 mm diameter and 100 mm height with the corner radius of 0.4 mm is to be produced by cup drawing. The required blank diameter is
- (a) 118 mm (b) 161 mm (c) 224 mm (d) 312 mm
57. A brass billet is to be extruded from its initial diameter of 100 mm to a final diameter of 50 mm. The working temperature of 700°C and the extrusion constant is 250 MPa. The force required for extrusion is
- (a) 5.44 MN (b) 2.72 MN (c) 1.36 MN (d) 0.36 MN

63. Two machines of the same production rate are available for use. On machine 1, the fixed cost is Rs.100 and the variable cost is Rs.2 per piece produced. The corresponding numbers for the machine 2 are Rs.200 and Re.1 respectively. The sale price of the first 800 units is Rs.3.50 per unit and subsequently it is only Rs.3.00. The breakeven production rate for each machine is
 (a) 75 (b) 100 (c) 150 (d) 600
64. A residential school stipulates the study hours as 8.00 pm to 10.30 pm. Warden makes random checks on a certain student 11 occasions a day during the study hours over a period of 10 days and observes that he is studying on 71 occasions. Using 95% confidence interval, the estimated minimum hours of his study during that 10 day period is
 (a) 8.5 hours (b) 13.9 hours (c) 16.1 hours (d) 18.4 hours
65. The sale of cycles in a shop in four consecutive months are given as 70, 68, 82, 95. Exponentially smoothing average method with a smoothing factor of 0.4 is used in forecasting. The expected number of sales in the next month is
 (a) 59 (b) 72 (c) 86 (d) 136
66. Market demand for springs is 8,00,000 per annum. A company purchases these springs in lots and sells them. The cost of making a purchase order is Rs.1,200. The cost of storage of springs is Rs.120 per stored piece per annum. The economic order quantity is
 (a) 400 (b) 2,828 (c) 4,000 (d) 8,000
67. A manufacturer produces two types of products, 1 and 2, at production levels of x_1 and x_2 respectively. The profit is given is $2x_1 + 5x_2$. The production constraints are
- $$\begin{aligned} x_1 + 3x_2 &\leq 40 \\ 3x_1 + x_2 &\leq 24 \\ x_1 + x_2 &\leq 10 \\ x_1 > 0, x_2 > 0 \end{aligned}$$
- The maximum profit which can meet the constraints is
 (a) 29 (b) 38 (c) 44 (d) 75

(a) P – 2 Q – 1 R – 7 S – 6

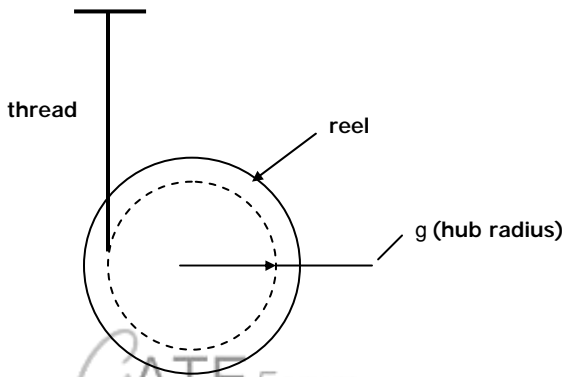
(b) P – 3 Q – 1 R – 5 S – 7

(c) P – 1 Q – 3 R – 1 S – 5

(d) P – 3 Q – 4 R – 7 S – 6

& ' (') * +! ,!* - +!

A reel of mass 'm' and radius of gyration 'k' is rolling down smoothly from rest with one end of the thread wound on it held in the ceiling as depicted in the figure. Consider the thickness of the thread and its mass negligible in comparison with radius 'r' of the hub and the reel mass 'm'. Symbol 'g' represents the acceleration due to gravity.



71. The linear acceleration of the reel is

(a) $\frac{2}{(2 + 2)}$

(b) $\frac{2}{(2 + 2)}$

(c) $\frac{2}{(2 + 2)}$

(d) $\frac{2}{(2 + 2)}$

72. The tension in the thread is

(a) $\frac{2}{(2 + 2)}$

(b) $\frac{2}{(2 + 2)}$

(c) $\frac{2}{(2 + 2)}$

(d) $\frac{2}{(2 + 2)}$

& ' (') * +! ,!* - +!

The state of stress at a point 'P' in a two dimensional loading is such that the Mohr's circle is a point located at 175 MPa on the positive normal stress axis.

73. Determine the maximum and minimum principal stresses respectively from the Mohr's circle

(a) + 175 MPa, -175 MPa

(b) + 175 MPa, -175 MPa

(c) 0, -175 MPa

(d) 0, 0

74. Determine the directions maximum and minimum principal stresses at the point 'P' from the Mohr's circle
 (a) $0, 90^\circ$ (b) $90^\circ, 0$ (c) $45^\circ, 135^\circ$ (d) all directions

& ' (' 0) * + ! , ! * - + ! " "

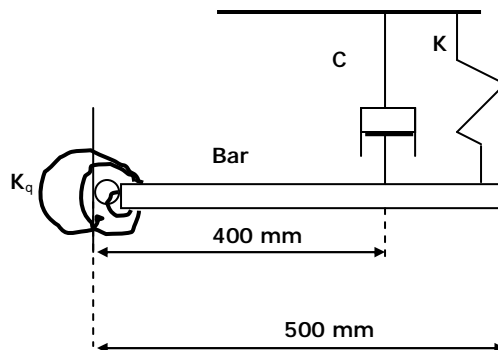
The circular disc shown in its plan view in the figure rotates in a plane parallel to the horizontal plane about the point O at a uniform angular velocity w . Two other points A and B are located on the line OZ at distances and from O respectively.

75. The velocity of point B with respect to point A is a vector of magnitude
 (a) 0
 (b) $w(-)$ and direction opposite to the direction of motion of point B
 (c) $w(-)$ and direction same as the direction of motion of point B
 (d) $w(-)$ and direction being from O to Z

76. The acceleration of point B with respect to point A is a vector of magnitude
 (a) 0
 (b) $w(-)$ and direction same as the direction of motion of point B
 (c) $w(-)$ and direction opposite to the direction of motion of point B
 (d) $w(-)$ and direction being from Z to O

& " (') * + ! , ! * - + ! " "

A uniform rigid slender bar of mass 10 kg, hinged at the left end is suspended with the help of spring and damper arrangement as shown in the figure where $K = 2\text{kN/m}$, $C = 500\text{ Ns/m}$ and the stiffness of the torsional spring K_q is 1 kN/m/rad . Ignore the hinge dimensions.



77. The un-damped natural frequency of oscillations of the bar about the hinge point is
 (a) 42.43 rad/s (b) 30 rad/s (c) 17.32 rad/s (d) 14.14 rad/s

& ' () * +! ,!* - +!

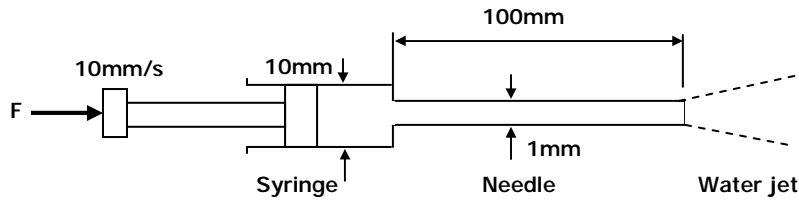
78. The damping coefficient in the vibration equation is given by
 (a) 500 Nms/rad (b) 500 N/(m/s) (c) 80 Nms/rad (d) 80 N/(m/s)

79. Z_2 and Z_4 are
 (a) 64 and 45 (b) 45 and 64 (c) 48 and 60 (d) 60 and 48

80. The center distance in the second stage is
 (a) 90 mm (b) 120 mm (c) 160 mm (d) 240 mm

& () * +! ,!* - +!

A syringe with a frictionless plunger contains water and has at its end a 100 mm long needle of 1 mm diameter. The internal diameter of the syringe is 10 mm. Water density is 1000 kg/m^3 . The plunger is pushed in at 10 mm/s and the water comes out as a jet.

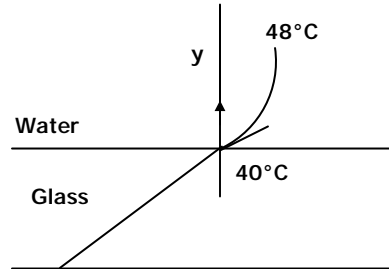


81. Assuming ideal flow, the force F in Newton's required on the plunger to push out the water is
 (a) 0 (b) 0.04 (c) 0.13 (d) 1.15

82. Neglect losses in the cylinder and assume fully developed laminar viscous flow throughout the needle; the Darcy friction factor is $64/Re$. Where Re is the Reynolds number. Given that the viscosity of water is $1.0 \times 10^{-3} \text{ kg/sm}$, the force F in newtons required on the plunger is
 (a) 0.13 (b) 0.16 (c) 0.3 (d) 4.4

& (.) * +! ,!* - +!

Heat is being transferred by convection from water at 48°C to a glass plate whose surface that is exposed to the water is at 40°C. The thermal conductivity of water is 0.6 W/mK and the thermal conductivity of glass is 1.2 W/mK. The spatial gradient of temperature in the water at the water-glass interface is $dT/dy = 1 \times 10^4$ K/m.



83. The value of the temperature gradient in the glass at the water-glass interface is K/m is
 (a) -2×10^4 (b) 0.0 (c) 0.5×10^4 (d) 2×10^4
84. The heat transfer coefficient h in $W/m^2 K$ is
 (a) 0.0 (b) 4.8 (c) 6 (d) 750

& / (0) * +! ,!* - +!

Nitrogen gas (molecular weight 28) is enclosed in a cylinder by a piston, at the initial condition of 2 bar, 298 K and $1 m^3$. In a particular process, the gas slowly expands under isothermal condition, until the volume becomes $2m^3$. Heat exchange occurs with the atmosphere at 298 K during this process.

85. The work interaction for the Nitrogen gas is
 (a) 200 kJ (b) 138.6 kJ (c) 2 kJ (d) -200 kJ
86. The entropy change for the Universe during the process in kJ/K is
 (a) 0.4652 (b) 0.0067 (c) 0 (d) -0.6711

& ' () * +! ,!* - +!

A refrigerator based on ideal vapour compression cycle operates between the temperature limits of $-20^\circ C$ and $40^\circ C$. The refrigerant enters the condenser as saturated vapour and leaves as saturated liquid. The enthalpy and entropy values for saturated liquid and vapour at these temperatures are given in the table below.

